CHAPTER 1

ENERGY ANALYSIS OF TWO STROKE ENGINES WITH A QUASI-REALISTIC THERMODYNAMIC MODEL APPROACH

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1. INTRODUCTION

As is known, internal combustion engines are based on the principle of obtaining high pressure in the cylinder by burning the fuel-air mixture. An internal combustion engine is formed by the sequential repetition of successive processes, that is, by the continuous occurrence of cycles. These processes are the intake of the mixture (air or air-fuel) into the cylinder (suction process), the compression of this mixture (compression process), the combustion of the compressed mixture, followed by power generation (expansion process), and the exhausting of the burned gases (exhaust process) (Figure 1). In internal combustion engines, these processes can take place in a single volume (for example, piston engines), or each process can take place in separate components (for example, gas turbines). There is also a 6-stroke engine concept that utilizes exhaust waste heat by performing two more processes after these processes [1-3].



Figure 1. Processes occurring in internal combustion engine cycles

The process in Figure 1 is independent on fuel. While a controlled ignition occurs in gasoline-powered engines, self-ignition oin diesel-powered engines by

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NOMENCLATURE

\dot{O}_{f}	power from fuel
$\dot{O}_{_{in}}$	input heat flow
\dot{O}_{out}	output heat flow
$\dot{m}_{_i}$	mass flow of fluid i
P _e	effective power
P_{i}	indicated power
P_{μ}	friction loss power
m _i	mass of fluid i
ртr	transfer pressure
S _p	mean piston speed
θE_{g}	exhaust closing angle
BDC _e	effectie bottom dead center
BDC _g	geometric bottom dead center
h	stroke
В	cylinder radius
Ν	engine speed
r	crank radius
λ	connecting rod-crank ratio
μ	coefficient of friction
\mathcal{E}_{e}	effective compression ratio
\mathcal{E}_{g}	geometric compression ratio

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